

08/806,031
In re ARHAB, et al.

IN THE SPECIFICATION

Please amend the specification as follows.

Page 1, lines 1 and 2, please delete in their entirety ("INTERNATIONAL PATENT
APPLICATION NO. PCT/FR00/02157, PUBLICATION NO. WO01/07801A1").

Page 1, between lines 2 and 3 (before the first paragraph), please insert the following
headings:

B2
BACKGROUND OF THE INVENTION

1. Field of the Invention

Page 1, between lines 4 and 5, please insert the following heading:

2. Description of Related Art

Page 2, between lines 3 and 4, insert and center the following heading:

SUMMARY OF THE INVENTION

Page 3, between lines 24 and 25, insert and center the following heading:

BRIEF DESCRIPTION OF THE DRAWINGS

Page 4, between lines 31 and 32, insert and center the following heading:

DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

Page 5, line 10, please amend (change "12" to --10--) as follows.

The wall 3 is extended at its outer periphery by an axially oriented annular first flange 6, to define a first casing element 3, 6. An axially oriented annular second flange 7 is centred in the region of its free edge on the free end of the first flange 6. This second flange 7 is therefore in intimate contact through its inner periphery with the outer periphery of the first flange 6, and in this example it is welded on the first flange 6. The second flange 7 is extended by a semi-toroidal envelope 8, on the inside of which the vanes 9 of an impulse wheel 10, which are in facing relationship with the vanes 11 of a turbine wheel 12, are fixed. The flange 7 and envelope 8 are part of a second casing element 7, 8. The piston 4 lies between the turbine wheel and the wall 3.

Page 6, line 35, please amend (change "14" (first occurrence) to --10--) as follows.

The skirt 27 stiffens the piston 4. In all cases the piston 4, radially inwardly of its surface 2, matches the form of the turbine wheel 12 and hub 14, so as to reduce the overall size of the hydrokinetic coupling apparatus, which comprises the turbine wheel 12, impulse wheel 10, piston 4, hub 14, and a torsion damper 28 which will be described below. The apparatus has an axial axis of symmetry and rotation X-X'. In this example, the apparatus also includes a reaction wheel 29 so that it constitutes a torque converter in the known way.

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Page 7, line 24, please amend (change "14" to --10--) as follows.

When the disc 31 is gripped, the lock-up clutch, which comprises the surfaces 1, 2 and the torsion damper 28, is said to be engaged or bridged, in such a way that the rotary driving motion is transmitted directly from the driving shaft - the crankshaft of a motor vehicle in the case of application to a motor vehicle - to the driven shaft through the lock-up clutch without any relative sliding between the turbine wheel 12 and impulse wheel 10, and this reduces the fuel consumption of the vehicle.

Page 7, line 26, please amend (change "13" to --31--) as follows.

When the disc 31 is released, the lock-up clutch is said to be disengaged or unbridged, so that the rotary driving motion is transmitted from the driving shaft to the driven shaft through the torque converter by virtue of the flow of oil between the vanes 9, 11 of the impulse wheel 10 and turbine wheel 12. This is what happens when the motor vehicle is started.

Page 8, line 16, please amend (change "41" to --40--) as follows.

The ring portions 55, 155 are in contact with each other. The damper plate 35 and the guide rings 36, 37 are provided with windows 38, 39, facing each other for holding resilient members 40 which in this example consist of coil springs or concentric helical springs. An axially acting spring 40 bears on the guide ring 37 which is proximal to the wall 3, for action on the damper plate 35 and for gripping the latter in contact with the other guide ring 36 which is proximal to the piston 4.

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Page 15, line 28, please amend (insert "1068" after --internal bore--) as follows.

In Figures 41 and 42, the bead of the projecting portion 1066 is not necessarily in the form of a collar portion, but may be formed in a rounded profile as visible at 1067 in Figure 42, and the internal bore 1068 of the passage hole formed in the ring 60 for receiving the projecting portion 1066 can have a central point such that fitting of the ring 60 is achieved by applying a pressure on the ring 60, leading to an eversion, that is to say one particular form of snap-fitting.

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